



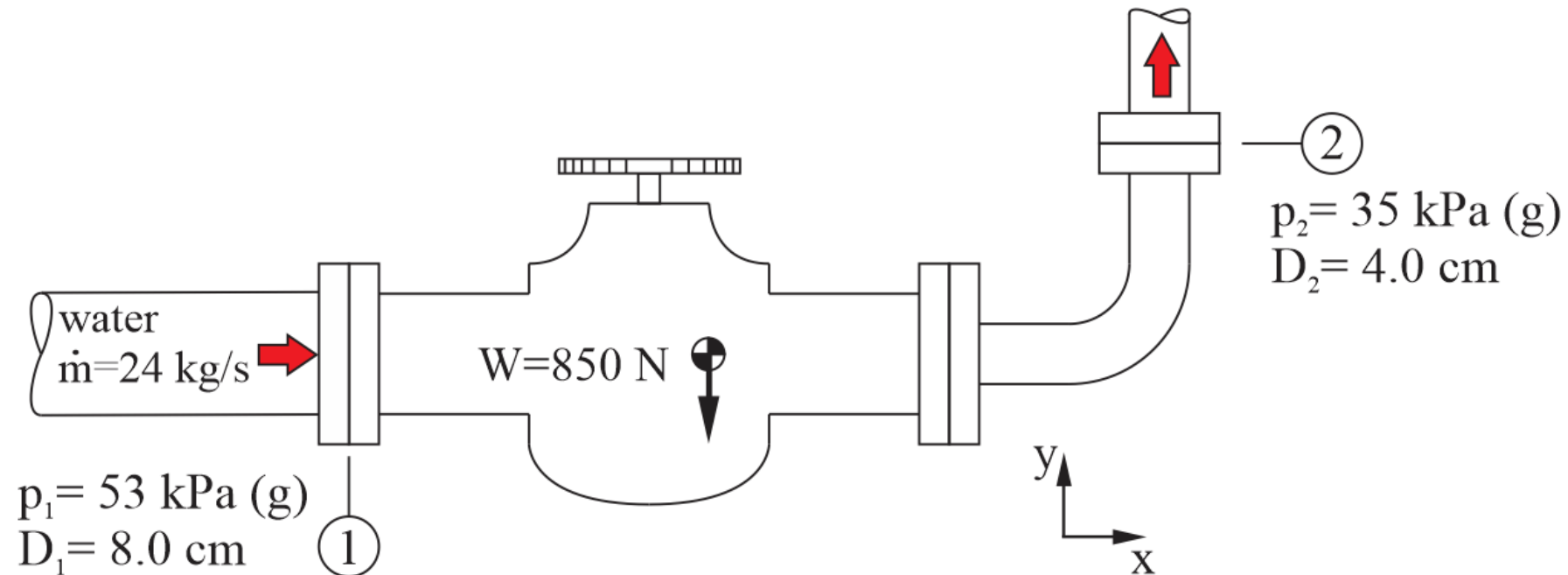
*MEC516/BME516:
Fluid Mechanics I*

*Chapter 3: Control Volume Analysis
Part 6.1: Example Problem*

Example: Linear Momentum (Makeup Final Exam 2017)

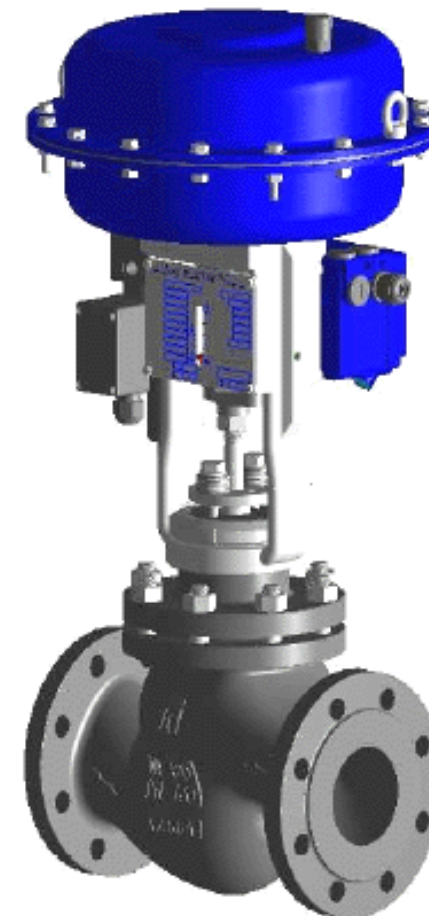
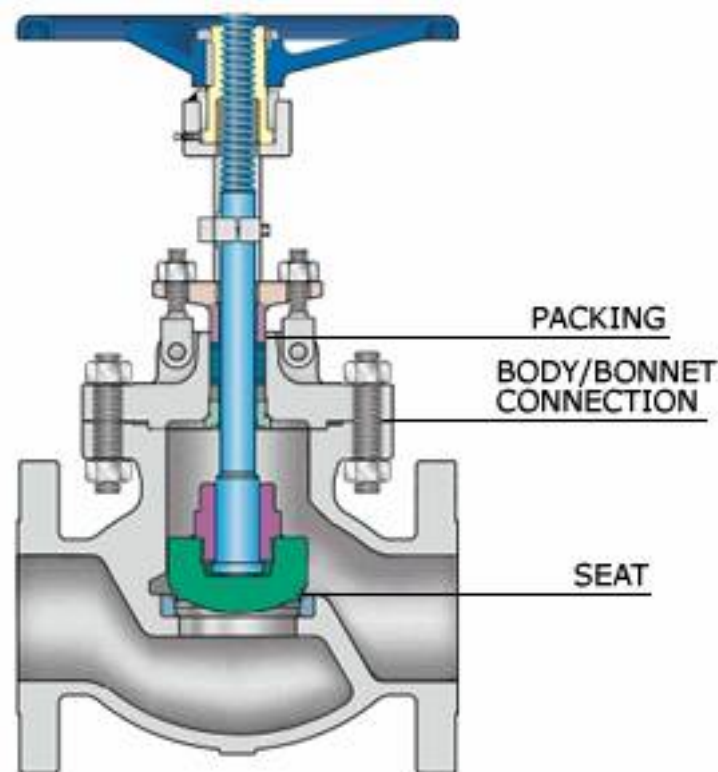
Water ($\rho = 998 \text{ kg/m}^3$) enters a valve at a mass flow rate of 24 kg/s . A *globe valve* reduces the pressure from 53 kPa (gauge) at the inlet to 35 kPa (gauge) at the outlet. The weight of the valve, pipe elbow, and the water from ① to ② is 850 N .

Calculate the forces in the x-direction (R_x) and y-direction (R_y) required to hold this valve and piping system in place.



An Aside: What is a Globe Valve?

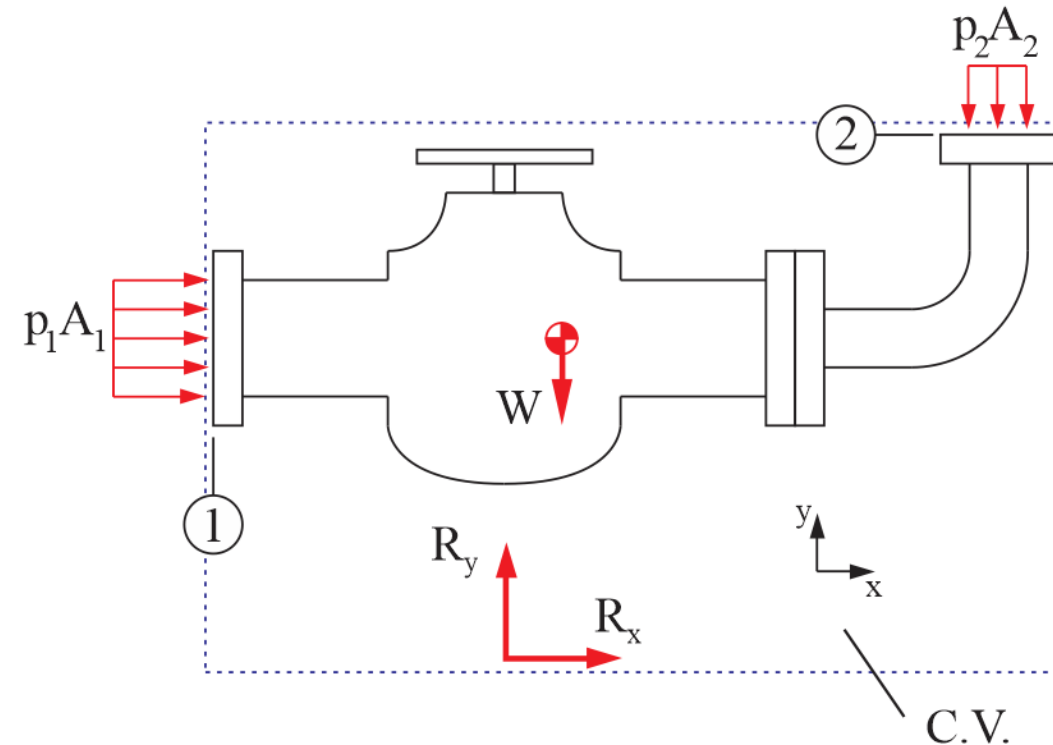
- Used for flow control in industrial piping systems
 - Hand wheel or pneumatic actuator for remote flow adjustment from control room



Example: Linear Momentum

Free Body Diagram

- Draw the isolated section. Work in symbols!
- All forces applied to C.V.:
 - Unknown reaction forces (R_x , R_y)
 - Weight(s)
 - Pressure forces, acting inward & normal to surface
 - Use **gauge** pressures. (Why?)



- Conservation of linear momentum. Steady flow, control volume with one inlet and one outlet:

$$\text{Continuity } \dot{m}_1 = \dot{m}_2 = \dot{m}$$

$$\sum \mathbf{F} = \dot{m}_2 \mathbf{V}_2 - \dot{m}_1 \mathbf{V}_1 = \dot{m}(\mathbf{V}_2 - \mathbf{V}_1)$$

All forces
on C.V.

Rate of momentum:
outflow – inflow

Example: Linear Momentum

- Vector equation: $\sum \mathbf{F} = \dot{m}(\mathbf{V}_2 - \mathbf{V}_1)$

$$\mathbf{F} = F_x \mathbf{i} + F_y \mathbf{j} \quad \mathbf{V} = u \mathbf{i} + v \mathbf{j}$$

x-direction: \rightarrow

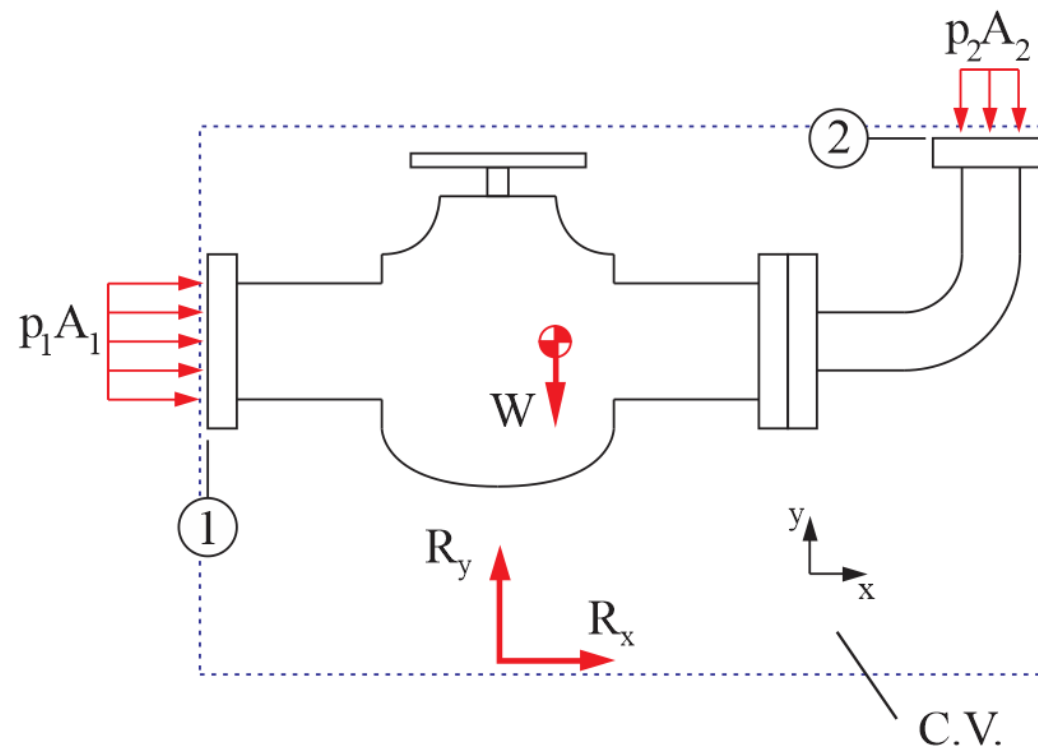
$$\sum F_x = \dot{m}(u_2 - u_1)$$

$$p_1 A_1 + R_x = \dot{m}(u_2 - u_1)$$

$u_2 = 0$ (no x-component at outlet)

Solve for R_x : $R_x = -p_1 A_1 - \dot{m}u_1$

- Force directions make sense?



Example: Linear Momentum

$$R_x = -p_1 A_1 - \dot{m} u_1$$

- Calculations:

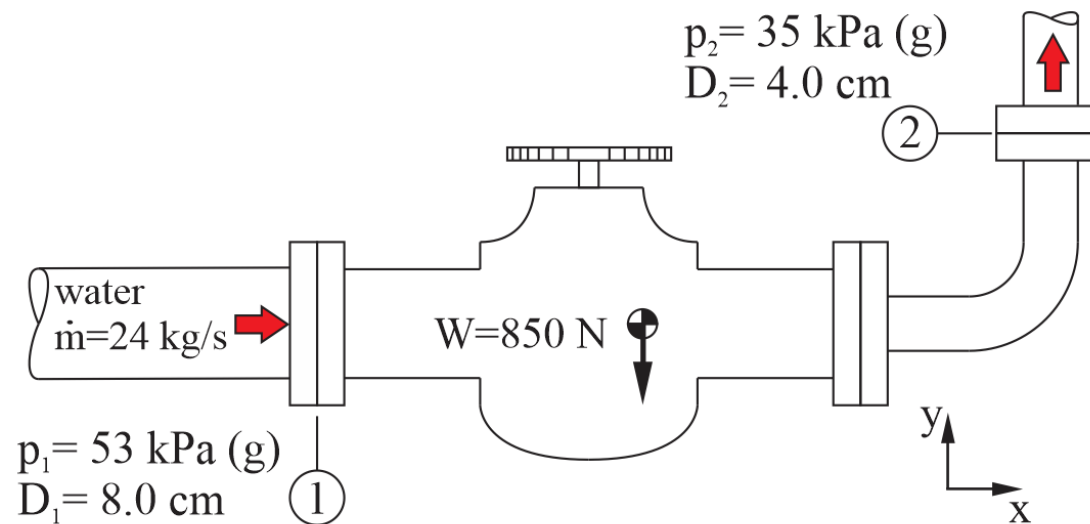
$$A_1 = \frac{\pi D_1^2}{4} = \frac{\pi (0.08\text{m})^2}{4} = 5.03 \times 10^{-3} \text{m}^2$$

$$\dot{m} = \rho A_1 u_1 \quad u_1 = \frac{\dot{m}}{\rho A_1}$$

$$u_1 = \frac{24 \text{ kg/s}}{998 \text{ kg/m}^3 (5.03 \times 10^{-3} \text{m}^2)} = 4.78 \text{ m/s}$$

$$R_x = -53 \times 10^3 \frac{\text{N}}{\text{m}^2} (5.03 \times 10^{-3} \text{m}^2) - 24 \frac{\text{kg}}{\text{s}} \left(4.78 \frac{\text{m}}{\text{s}} \right) = -266.4 - 114.8 \text{ N} = -381 \text{ N}$$

$$R_x = 381 \text{ N} \leftarrow \text{Ans.}$$



Example: Linear Momentum

Vector equation: $\sum \mathbf{F} = \dot{m}(\mathbf{V}_2 - \mathbf{V}_1)$

y-direction: \uparrow

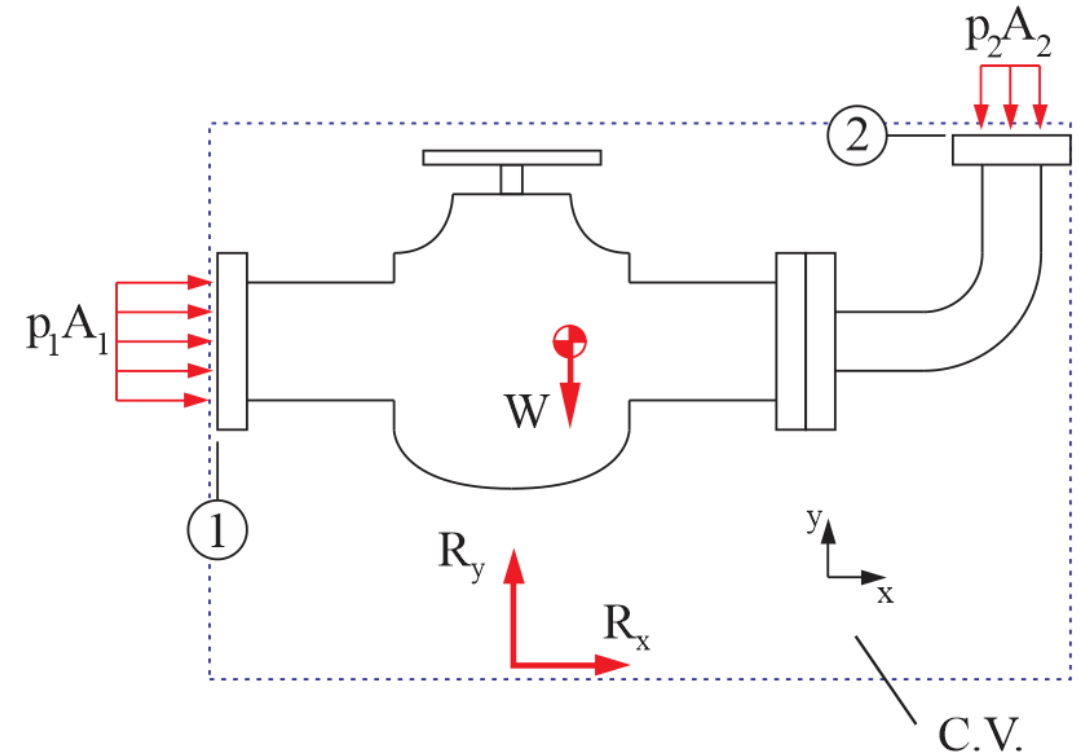
$$\sum F_y = \dot{m}(v_2 - v_1)$$

$v_1 = 0$ (no y-component at inlet)

$$-p_2A_2 - W + R_y = \dot{m}(v_2 - \cancel{v_1})$$

Solve for R_y : $R_y = p_2A_2 + W + \dot{m}v_2$

- Force directions make sense?



Example: Linear Momentum

$$R_y = p_2 A_2 + W + \dot{m} v_2$$

Calculations:

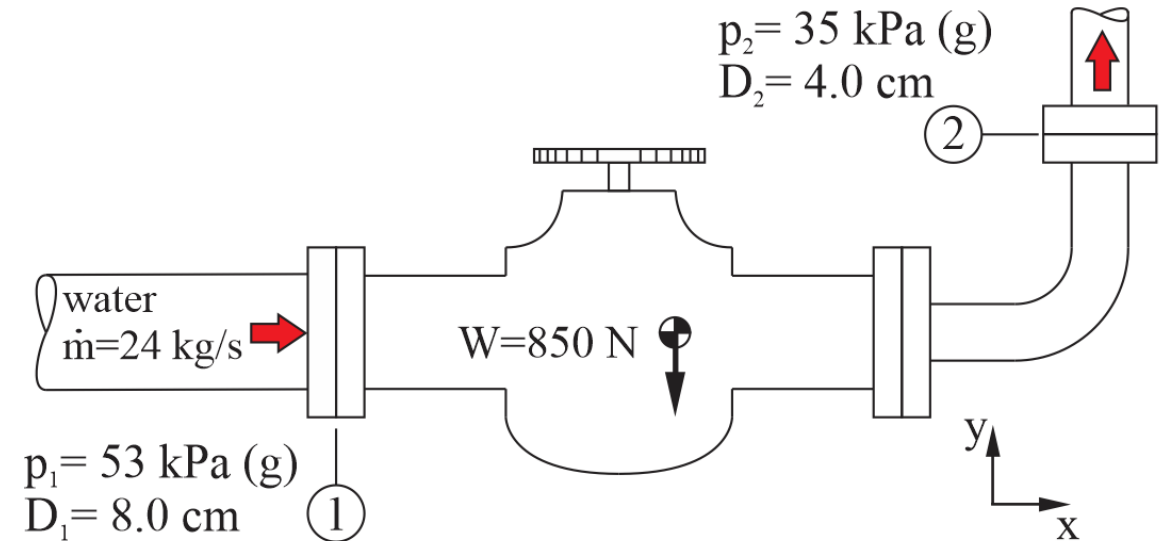
$$A_2 = \frac{\pi D_2^2}{4} = \frac{\pi (0.04 \text{ m})^2}{4} = 1.26 \times 10^{-3} \text{ m}^2$$

$$\dot{m} = \rho A_2 v_2 \quad v_2 = \frac{\dot{m}}{\rho A_2}$$

$$v_2 = \frac{24 \text{ kg/s}}{998 \text{ kg/m}^3 (1.26 \times 10^{-3} \text{ m}^2)} = 19.1 \text{ m/s}$$

$$R_y = 35 \times 10^3 \frac{\text{N}}{\text{m}^2} (1.26 \times 10^{-3} \text{ m}^2) + 850 \text{ N} + 24 \frac{\text{kg}}{\text{s}} \left(19.1 \frac{\text{m}}{\text{s}} \right) = +1350 \text{ N}$$

$$R_y = 1350 \text{ N} \uparrow \text{ Ans.}$$

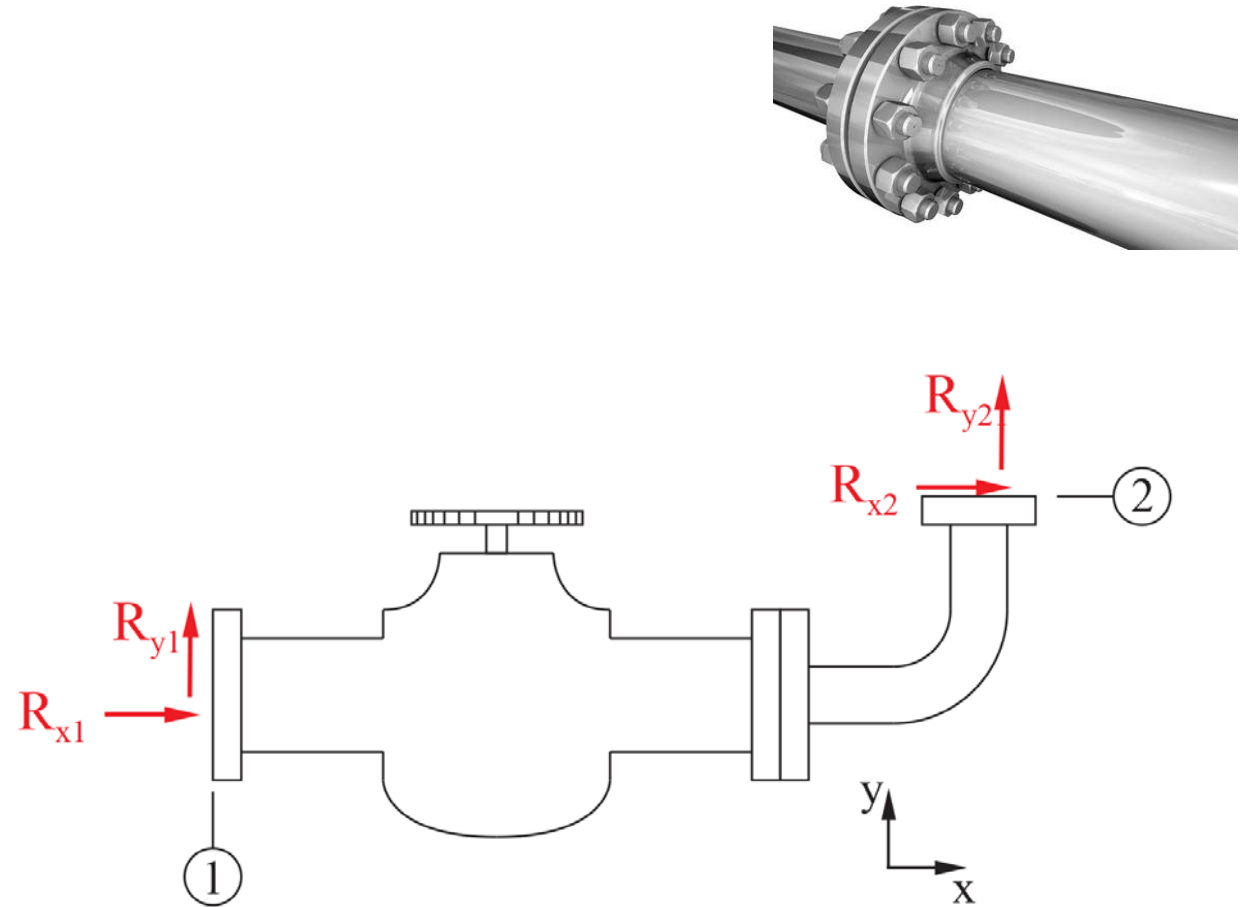


Example: Linear Momentum

- R_x and R_y act at the pipe flanges
- R_x and R_y are the net forces required to hold valve and pipe in place
- Linear momentum analysis can only determine *total* force in each direction:

$$R_x = R_{x1} + R_{x2} = -381 \text{ N}$$

$$R_y = R_{y1} + R_{y2} = 1350 \text{ N}$$



$$R_x = 381 \text{ N } \leftarrow$$

$$R_y = 1350 \text{ N } \uparrow$$



Streamlines/Streaklines over a Car

Source: <http://6speedhaven.tumblr.com/post/25448453293>

END NOTES

Presentation prepared and delivered by Dr. David Naylor.

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