



*MEC516/BME516:
Fluid Mechanics I*

*Chapter 3: Control Volume Analysis
Part 5*

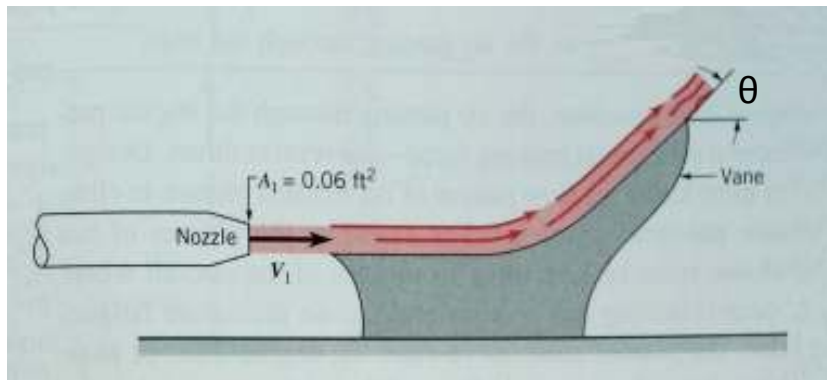
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Overview

- Conservation of Momentum for a Control Volume
 - use RTT to derive the “Linear Momentum Equation”
- Numerical Example
 - Calculating fluid forces on a vane. Has applications to rotation machinery, e.g. Pelton Wheel Turbine



http://commons.wikimedia.org/wiki/File:Pelton_wheel_turbine_in_Barcelona.jpg

Derivation of the Linear Momentum Equation

- Newton's second law, $\Sigma \mathbf{F} = m\mathbf{a} = \frac{d}{dt}(m\mathbf{V})_{sys}$

Sum of external forces = Rate of change of linear momentum of the system

- Note: Typeset **Bold** variables are vectors (textbook notation), HAND WRITTEN \vec{V}
- For a non-accelerating control volume, with 1-D flow normal to the control surface boundaries, RTT gives:

$$\frac{d}{dt}(B_{SYS}) = \frac{d}{dt} \int_{CV} \beta \, dm + \sum_{outlets} \beta_o \dot{m}_o - \sum_{inlets} \beta_i \dot{m}_i$$

(relates the system to the c.v.)

- The quantity of interest is $\mathbf{B} = m\mathbf{V}$, linear momentum. So,

$$\beta = \frac{d\mathbf{B}}{dm} = \mathbf{V}$$

Derivation of the Linear Momentum Equation

So, we get:
$$\frac{d}{dt}(m\mathbf{V})_{sys} = \frac{d}{dt} \int_{CV} \mathbf{V} dm + \sum_{outlets} \mathbf{V}_o \dot{m}_o - \sum_{inlets} \mathbf{V}_i \dot{m}_i$$

- Now, applying Newton's second law to the system:

$$\sum F = \frac{d}{dt}(m\mathbf{V})_{sys}$$

$$\sum F = \frac{d}{dt} \int_{CV} \mathbf{V} dm + \sum_{outlets} \mathbf{V}_o \dot{m}_o - \sum_{inlets} \mathbf{V}_i \dot{m}_i$$

In this course we will be dealing mostly with steady flows:

$$\sum F = \sum_{outlets} \dot{m}_o \mathbf{V}_o - \sum_{inlets} \dot{m}_i \mathbf{V}_i$$

Linear Momentum Equation

For steady 1-D flows (in a non-accelerating reference frame):

$$\sum F = \sum_{outlets} \dot{m}_o \mathbf{V}_o - \sum_{inlets} \dot{m}_i \mathbf{V}_i$$

Important Notes:

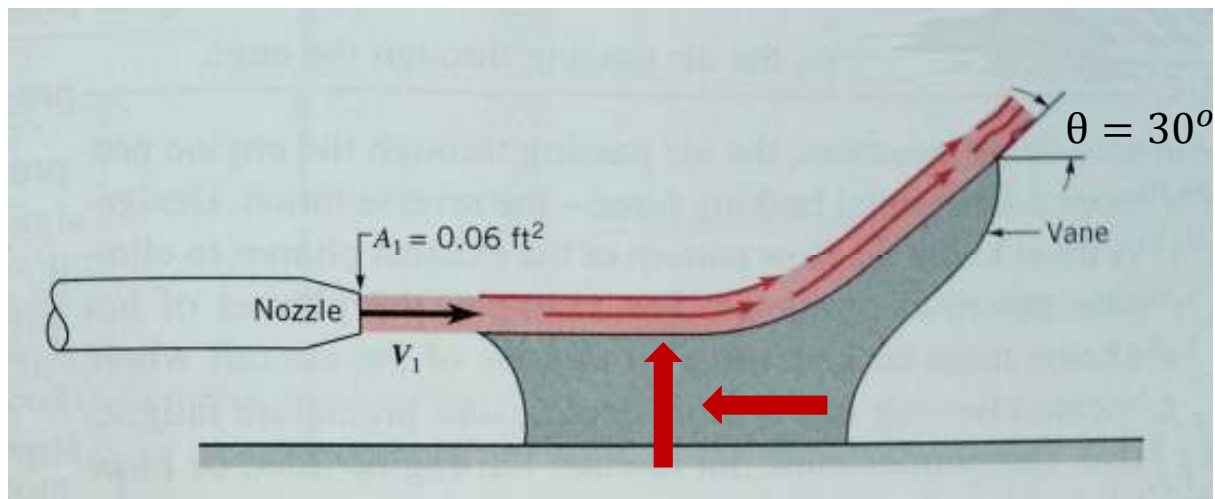
- \mathbf{V} is a vector quantity.
- ΣF is the vector sum of all forces acting on the system, including
 - all forces at the control surfaces: pressure and viscous forces on the fluid & forces in solids cut by the c.s.
 - weight of fluid (or solid) contained in the system
- For a moving control volume, \mathbf{V} is the relative velocity, i.e., $\mathbf{V} = \mathbf{V}_r$.

Example: Fluid forces on a stationary vane

A horizontal jet of water (at 50°F) exits a nozzle with a steady uniform velocity of 10 ft/s. The outlet area of the nozzle is 0.06 ft². A stationary vane redirects the jet upward, through an angle of $\theta=30^\circ$. Gravity and viscous forces are assumed to be negligible.

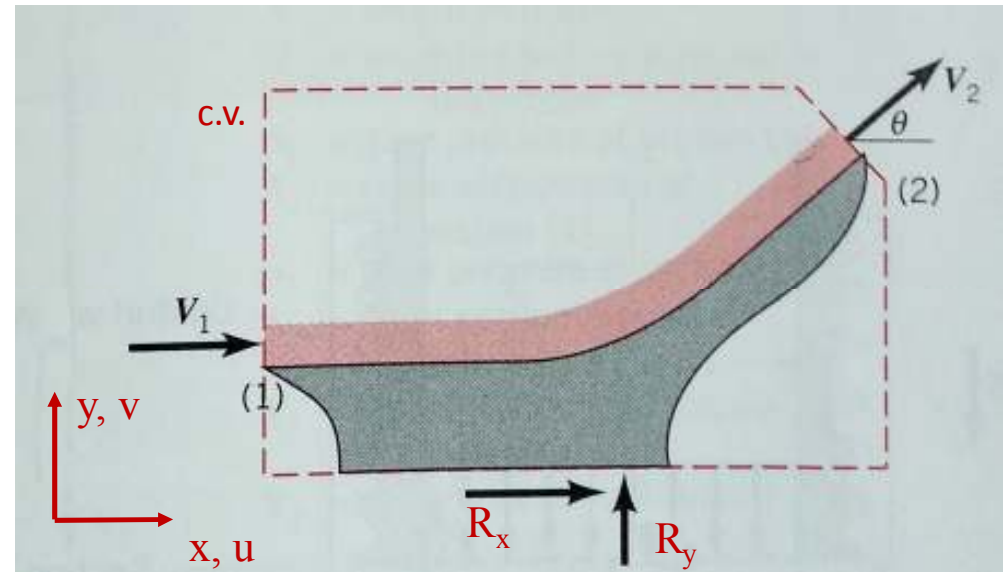
Calculate the force required to hold the vane stationary.

(What is your intuition?)



Example

- Control volume cuts through fluid & solid support.
- Free body diagram with the support reaction forces, R_x and R_y .
- The pressure at section (1) and (2) are the same, atmospheric pressure.
- With negligible gravity and viscous effect, and since $p_1=p_2$, the speed of the jet remains constant.



$$|\mathbf{V}_1| = |\mathbf{V}_2| = V = 10 \text{ ft/s, also } \dot{m}_1 = \dot{m}_2 = \dot{m} = \rho AV$$

- For steady flow, the linear momentum equation with one inlet (1) and one outlet (2) is:

$$\sum \mathbf{F} = \dot{m}_2 \mathbf{V}_2 - \dot{m}_1 \mathbf{V}_1 = \dot{m} \mathbf{V}_2 - \dot{m} \mathbf{V}_1$$

Example

- Keeping in mind the vector nature of the forces and velocities, we get:

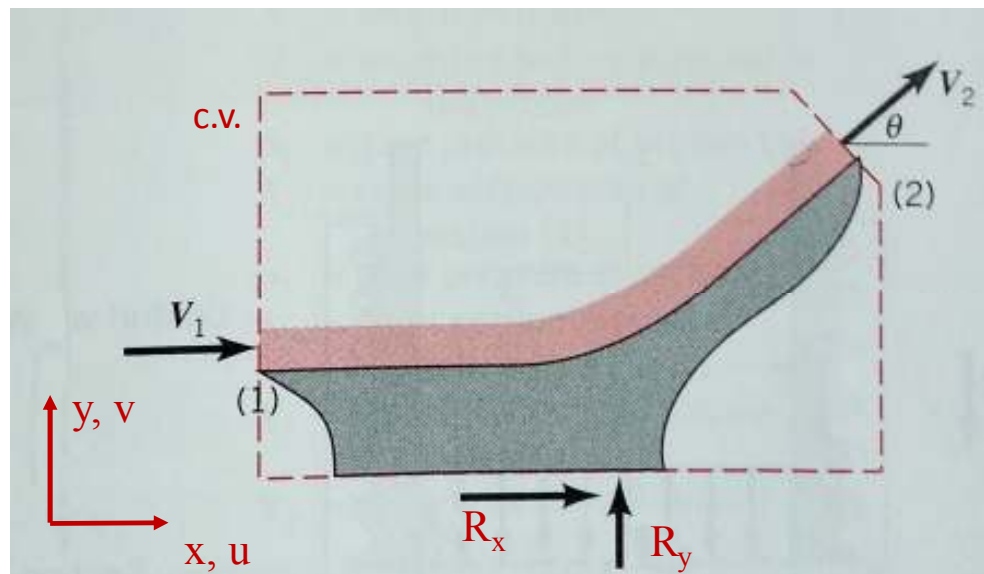
$$\mathbf{V}_1 = u_1 \mathbf{i} + v_1 \mathbf{j}; \quad u_1 = V = 10 \frac{ft}{s}, \quad v_1 = 0$$

$$\mathbf{V}_2 = u_2 \mathbf{i} + v_2 \mathbf{j}; \quad u_2 = V \cos \theta, \quad v_2 = V \sin \theta$$

So,

$$\Sigma F_x = R_x = \dot{m}(u_2 - u_1) = \dot{m}(V \cos \theta - V) = \dot{m}V (\cos \theta - 1)$$

$$\Sigma F_y = R_y = \dot{m}(v_2 - v_1) = \dot{m}V \sin \theta$$



Example

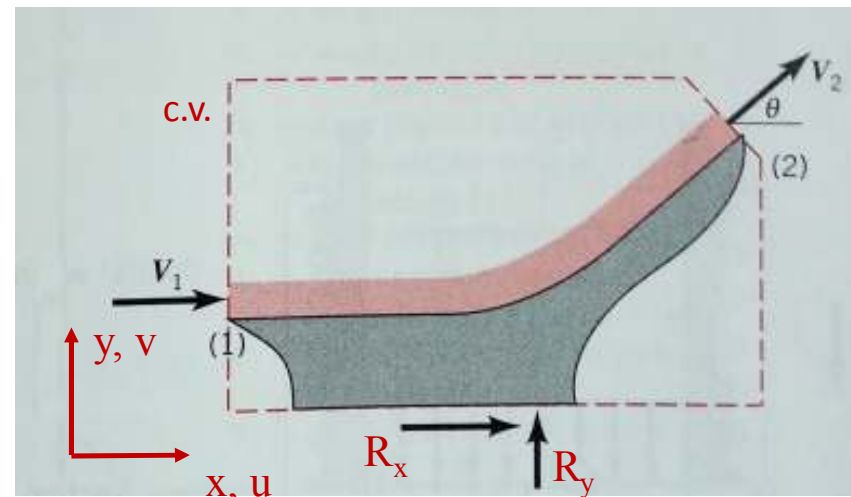
Liquid water (at 50°F), $\rho = 1.94 \frac{\text{slugs}}{\text{ft}^3}$

$$\begin{aligned} \dot{m} &= \rho A_1 V = 1.94 \frac{\text{slugs}}{\text{ft}^3} (0.06 \text{ft}^2) \left(10 \frac{\text{ft}}{\text{s}}\right) \\ &= 1.164 \text{ slugs/s} \end{aligned}$$

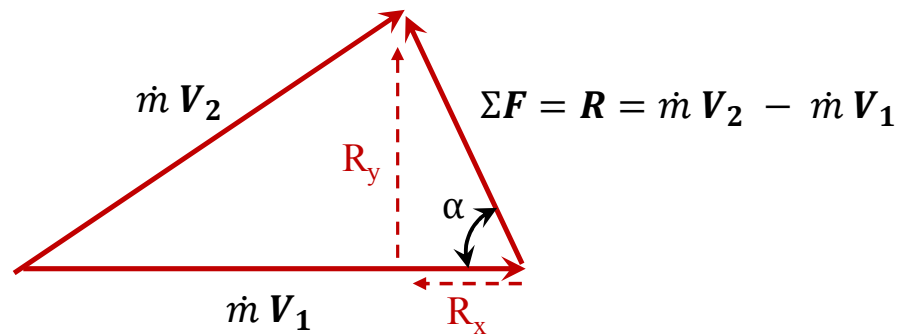
$$R_x = \dot{m}V (\cos \theta - 1) = 1.164 \frac{\text{slugs}}{\text{s}} \left(10 \frac{\text{ft}}{\text{s}}\right) (\cos 30^\circ - 1) = -1.559 \text{ lb} \quad (R_x = +1.559 \text{ lb} \leftarrow)$$

$$R_y = \dot{m}V \sin \theta = 1.164 \frac{\text{slugs}}{\text{s}} \left(10 \frac{\text{ft}}{\text{s}}\right) \sin 30^\circ = 5.820 \text{ lb} \uparrow$$

$$\text{Magnitude of Total Reaction: } R = \sqrt{R_x^2 + R_y^2}$$



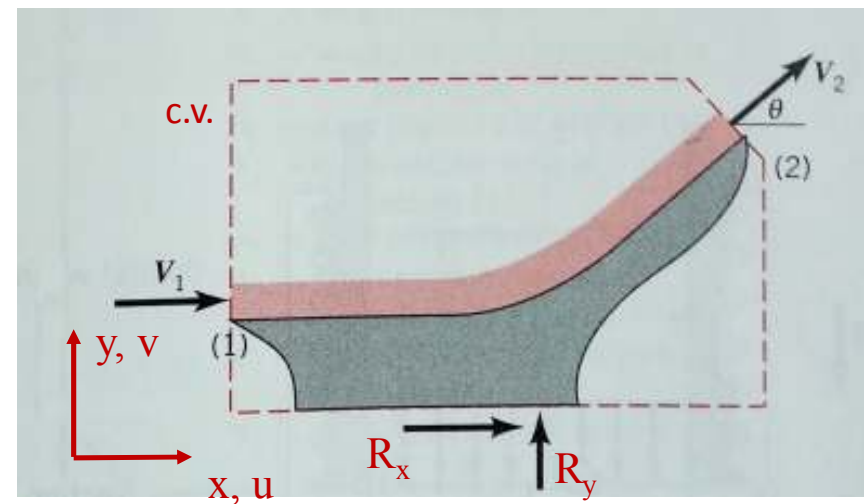
Example

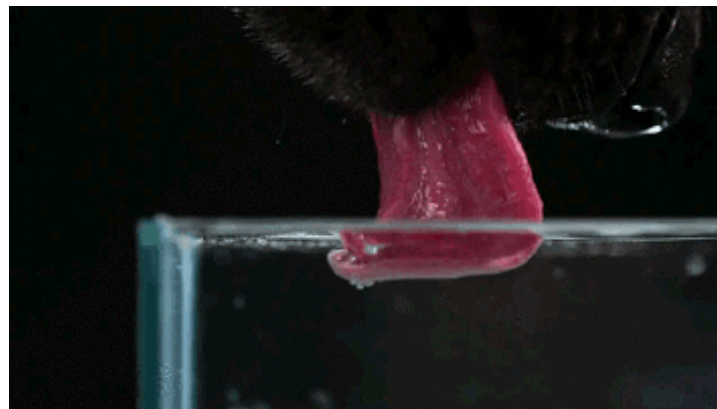


$$\text{Total Reaction } R = \sqrt{R_x^2 + R_y^2} = (1.56^2 + 5.82^2)^{1/2}$$

$$R = 6.02 \text{ lb} \quad \alpha = \tan^{-1} \frac{R_y}{R_x} = 75.0^\circ$$

Ans.





How a Dog Drinks

Source: <http://thesirens-of-titan.tumblr.com/post/80516755506/this-high-speed-footage-shows-how-a-dog-drinks>

END NOTES

Presentation prepared and delivered by Dr. David Naylor.

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